

Recalls of Automatic Control

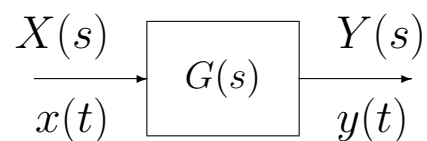
- The physical systems are often mathematically described by linear time-invariant differential equations. Example:

$$\ddot{y}(t) + 3\dot{y}(t) + 2y(t) = \dot{x}(t) + 5x(t)$$

- In the Laplace transformed space the solution $Y(s)$ of the differential equation, when the initial conditions are zero, is the product of the transfer function $G(s)$ and the Laplace transform $X(s)$ of the input signal $x(t)$:

$$Y(s) = \underbrace{\frac{(s+5)}{s^2+3s+2}}_{G(s)} X(s) = G(s) X(s)$$

- The system can be graphically described by a SISO (Single Input Single Output) block scheme:



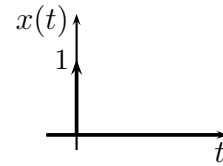
- The time solution $y(t)$ of the given differential equation can be obtained using the inverse Laplace transform:

$$y(t) = \mathcal{L}^{-1}[Y(s)] = \mathcal{L}^{-1}[G(s)X(s)]$$

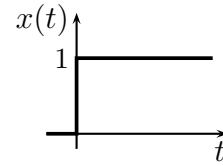
- The Laplace approach:
 - is powerful and quite simple, but only the input and output signals $x(t)$ and $y(t)$ of the given system are considered.
 - does not show the time behavior of the internal variables of the system.
 - is not suitable for describing some “particular” linear systems.
 - is not suitable for describing nonlinear systems.

• Laplace transform of the main time signals:

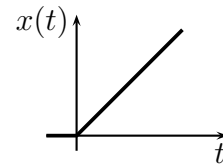
$$x(t) = \delta(t) \quad \leftrightarrow \quad X(s) = 1$$



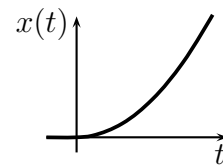
$$x(t) = u(t) \quad \leftrightarrow \quad X(s) = \frac{1}{s}$$



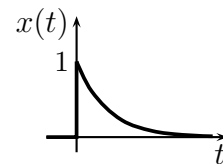
$$x(t) = t \quad \leftrightarrow \quad X(s) = \frac{1}{s^2}$$



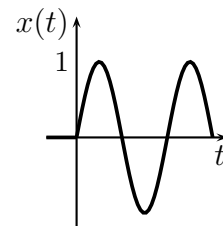
$$x(t) = \frac{t^2}{2} \quad \leftrightarrow \quad X(s) = \frac{1}{s^3}$$



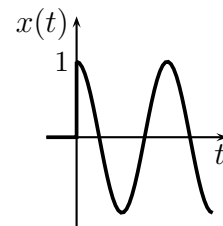
$$x(t) = e^{-at} \quad \leftrightarrow \quad X(s) = \frac{1}{s+a}$$



$$x(t) = \sin \omega t \quad \leftrightarrow \quad X(s) = \frac{\omega}{s^2 + \omega^2}$$



$$x(t) = \cos \omega t \quad \leftrightarrow \quad X(s) = \frac{s}{s^2 + \omega^2}$$



$$x(t) = t^n e^{-at} \quad \leftrightarrow \quad X(s) = \frac{n!}{(s+a)^{n+1}}$$

$$x(t) = e^{-at} \sin \omega t \quad \leftrightarrow \quad X(s) = \frac{\omega}{(s+a)^2 + \omega^2}$$

$$x(t) = e^{-at} \cos \omega t \quad \leftrightarrow \quad X(s) = \frac{s+a}{(s+a)^2 + \omega^2}$$

• Main properties of the Laplace transform:

Linearity:

$$\mathcal{L}[c_1 x_1(t) + c_2 x_2(t)] = c_1 X_1(s) + c_2 X_2(s)$$

Time shift:

$$\mathcal{L}[x(t - t_0)] = e^{-t_0 s} X(s)$$

Time integral:

$$\mathcal{L}\left[\int_0^t x(\tau) d\tau\right] = \frac{1}{s} X(s)$$

Time derivative (x_0 is the initial condition):

$$\mathcal{L}\left[\frac{dx(t)}{dt}\right] = s X(s) - x_0$$

Initial value theorem:

$$\lim_{t \rightarrow 0^+} x(t) = \lim_{s \rightarrow \infty} s X(s)$$

Final value theorem (only if $X(s)$ is a stable function):

$$\lim_{t \rightarrow \infty} x(t) = \lim_{s \rightarrow 0} s X(s)$$

Multiplying an exponential function:

$$\mathcal{L}[e^{-at} x(t)] = X(s + a)$$

• Examples of Laplace transforms in Matlab:

```
-----
syms s t
xt=5+3*exp(-2*t)*sin(5*t);
Xs=laplace(xt,t,s);
pretty(Xs)
-->
      5      15
      - + -----
      s      (s + 2) + 25
-----
```

```
-----
syms s t
xt=4*cos(5*t)+4*t^2*exp(-3*t);
Xs=laplace(xt,t,s);
pretty(Xs)
-->
      8      4 s
      ----- + -----
      3      2
      (s + 3)  s + 25
-----
```

Partial fraction expansion method

- A rational function $Y(s)$ which has only *simple poles* can always be expressed as follows:

$$Y(s) = \frac{N(s)}{(s - p_1)(s - p_2) \dots (s - p_n)} = \sum_{i=1}^n \frac{K_i}{s - p_i}$$

where parameters K_i (called *residues*) can be obtained as follows:

$$K_i = (s - p_i)Y(s) \Big|_{s=p_i}$$

- So, the time function $y(t)$ can be expressed as follows:

$$y(t) = \sum_{i=1}^n K_i e^{p_i t}$$

- Examples of inverse Laplace transforms in Matlab:

```
-----
syms s t
Xs=(5*s+3)/((s+1)*(s+2)*(s+3));
xt=ilaplace(Xs,s,t);          -->  7 exp(-2 t) - exp(-t) - 6 exp(-3 t)
pretty(xt)
-----
```

```
-----
syms s t
Xs=(7*s^2-8*s+5)/(s^3+2*s^2+5*s);
xt=ilaplace(Xs,s,t);          -->  6 exp(-t) | cos(2 t) - 4 sin(2 t) | + 1
pretty(xt)                    \          3          /
-----
```

```
-----
syms s t
Xs=(5)/(s^3*(s+1)^2);
xt=ilaplace(Xs,s,t);          -->  ----- - 15 exp(-t) - 5 t exp(-t) - 10 t + 15
pretty(xt)                    2
-----
```

Step Response of First Order Systems

- Let us consider a linear first order differential equation:

$$a \dot{y}(t) + b y(t) = c x(t)$$

- The corresponding transfer function $G(s)$ is:

$$G(s) = \frac{c}{a s + b} = \frac{K_0}{(1 + \tau s)}$$

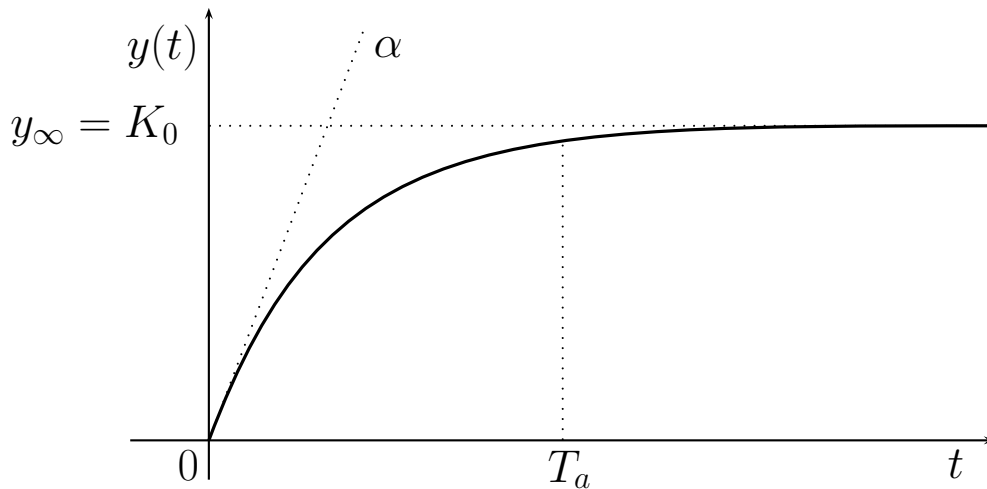
- The *constant time* τ and the *static gain* K_0 are defined as follows:

$$\tau = \frac{a}{b}, \quad K_0 = G(s)|_{s=0} = G(0) = \frac{c}{b}.$$

- When $x(t) = 1$, the step response $y(t)$ of system $G(s)$ is:

$$y(t) = \mathcal{L}^{-1} \left[G(s) \frac{1}{s} \right] = \mathcal{L}^{-1} \left[\frac{K_0}{s(1 + \tau s)} \right] = K_0 \left(1 - e^{-\frac{t}{\tau}} \right)$$

- The time behavior of function $y(t)$ is aperiodic and exponential:



- The *settling time* T_a is:

$$T_a = 3\tau = \frac{3}{|p|} = \frac{3}{\delta\omega_n}, \quad p = -\frac{1}{\tau}$$

- When $x(t) = 1$, the *final value* y_∞ is equal to the static gain K_0 :

$$y_\infty = \lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} s Y(s) = K_0 = \frac{c}{b}$$

- The *initial slope* α can be determined using the initial value theorem:

$$\alpha = \lim_{t \rightarrow 0^+} \dot{y}(t) = \lim_{s \rightarrow \infty} s(sY(s)) = \lim_{s \rightarrow \infty} sG(s) = \frac{c}{a}$$

Step Response of Second Order Systems

- Let us consider a linear second order differential equation:

$$a \ddot{y}(t) + b \dot{y}(t) + c y(t) = d x(t)$$

- The corresponding transfer function $G(s)$ is:

$$G(s) = \frac{d}{a s^2 + b s + c} = \frac{K_0}{\frac{s^2}{\omega_n^2} + 2\delta \frac{s}{\omega_n} + 1} = \frac{K_0 \omega_n^2}{s^2 + 2\delta \omega_n s + \omega_n^2}.$$

- The *natural frequency* ω_n , the *damping ratio* δ and the *static gain* K_0 are defined as follows:

$$\omega_n = \sqrt{\frac{c}{a}}, \quad \delta = \frac{b}{2\sqrt{ac}}, \quad K_0 = \frac{d}{c}.$$

- The step response $y(t)$ of system $G(s)$ is:

$$y(t) = \mathcal{L}^{-1} \left[\frac{K_0 \omega_n^2}{s(s^2 + 2\delta \omega_n s + \omega_n^2)} \right] = K_0 \left[1 - \frac{e^{-\sigma t}}{\sqrt{1 - \delta^2}} \text{sen}(\omega t + \varphi) \right].$$

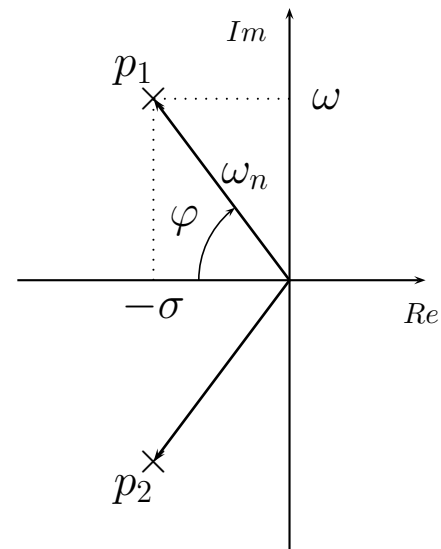
- The parameters $-\sigma$ and ω are the real and imaginary parts of the two complex conjugate poles $p_{1,2}$ of function $G(s)$:

$$p_{1,2} = -\sigma \pm j\omega$$

$$\sigma = \delta \omega_n = \frac{b}{2a}$$

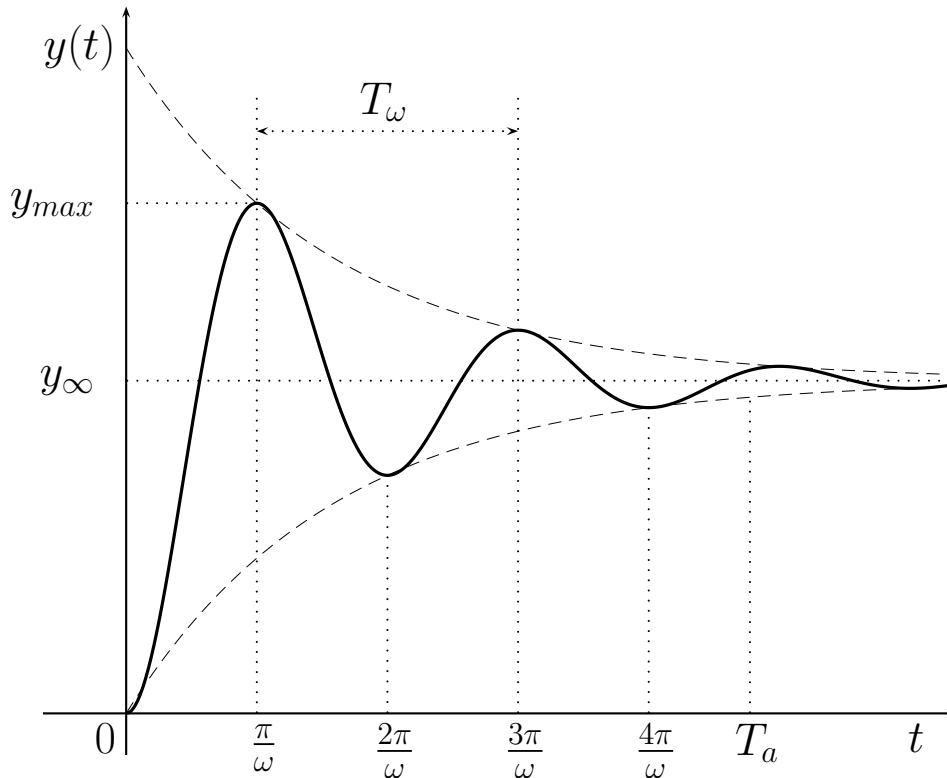
$$\omega = \omega_n \sqrt{1 - \delta^2} = \frac{\sqrt{4ac - b^2}}{2a}$$

$$\varphi = \arccos \delta = \arctan \frac{\sqrt{1 - \delta^2}}{\delta}$$



- This time analysis is valid only for $|\delta| \leq 1$. When $0 \leq \delta \leq 1$ the system is stable: $\sigma \geq 0$. When $-1 \leq \delta < 0$ the system is unstable: $\sigma < 0$.

- The time behavior of function $y(t)$ is exponential damped sinusoidal:



- The *settling time* T_a can be determined as follows:

$$T_a = \frac{3}{\sigma} = \frac{3}{\delta\omega_n},$$

- The *period* T_ω can be determined as follows:

$$T_\omega = \frac{2\pi}{\omega}$$

- The *percent overshoot* $S\%$ is defined as follows:

$$S\% = 100 \frac{(y_{max} - y_\infty)}{y_\infty}$$

- The *percent overshoot* $S\%$ can be determined as follows:

$$S\% = 100 e^{\frac{-\pi\delta}{\sqrt{1-\delta^2}}} = 100 e^{\frac{-\pi}{\tan\varphi}} = 100 e^{\frac{-\pi\omega}{\sigma}}$$

Step response of systems with one dominant pole

- Given the following system $G(s)$:

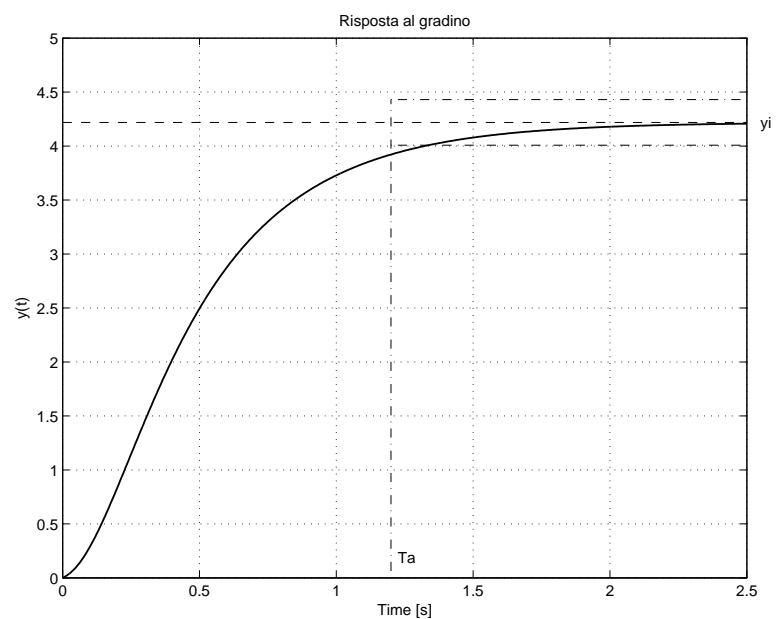
$$G(s) = \frac{(3 + 0.2s)(s^2 + 60s + 1800)}{\underbrace{(2 + 0.8s)}_{\text{polo dominante}}(8 + 0.2s)(s^2 + 16s + 80)}$$

- Dominant pole p_1 , final value y_∞ , settling time T_a and time behavior of function $G(s)$:

$$p_1 = G(0) = -2.5$$

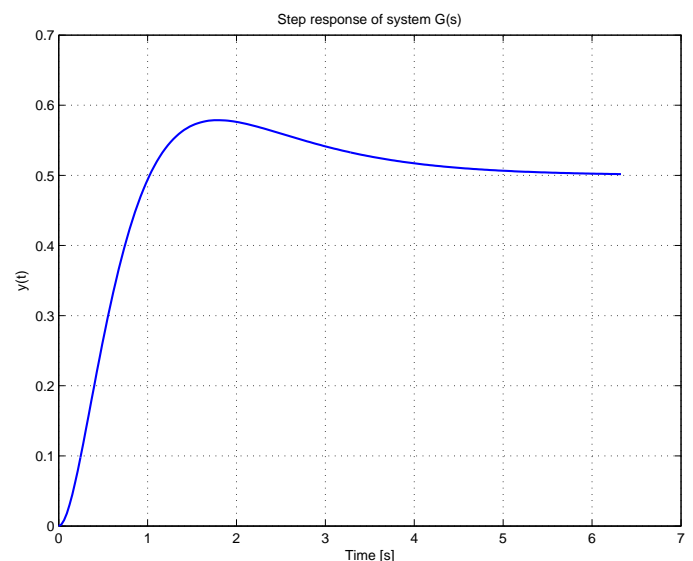
$$y_\infty = G(0) = 4.22$$

$$T_a \simeq \frac{3}{2.5} \text{ s} = 1.2 \text{ s},$$



- Examples of step response in Matlab:

```
-----
s=tf('s');
Gs=(5*s+3)/((s+1)*(s+2)*(s+3));
[yt,t]=step(Gs);
figure(1); clf
plot(t,yt,'Linewidth',1.5);
grid on
title('Step response of system G(s)')
ylabel('y(t)')
xlabel('Time [s]')
-----
```



Step response of systems with two dominant poles

- Given the following system $G(s)$:

$$G(s) = \frac{800(2s + 30)}{(0.2s + 3)(2s + 10) \underbrace{(s^2 + s + 100)}_{\text{poli dominanti}}(s^2 + 20s + 400)}$$

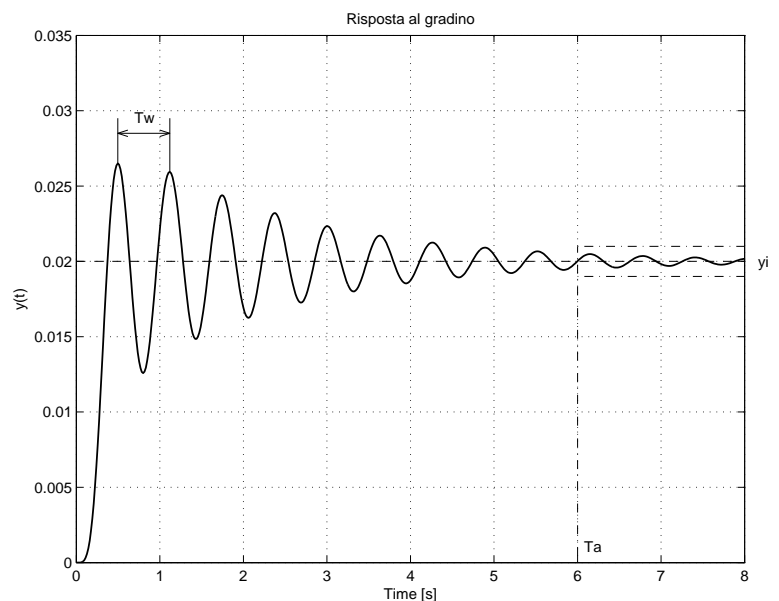
- Dominant poles $p_{1,2}$, final value y_∞ , settling time T_a , period T_w and time behavior of function $G(s)$:

$$p_{1,2} \simeq -0.5 \pm j 10$$

$$y_\infty = G(0) = 0.02.$$

$$T_a \simeq \frac{3}{0.5} \text{ s} = 6 \text{ s}$$

$$T_w \simeq \frac{2\pi}{10} \text{ s} = 0.63 \text{ s}$$



- Examples of step response in Matlab:

```
-----
s=tf('s');
Gs=(5*s+30)/((s^2+2*s+15)*(0.2*s+4)*(0.1*s+2));
[yt,t]=step(Gs,5);
figure(1); clf
plot(t,yt,'Linewidth',1.5);
grid on
title('Step response of system G(s)')
ylabel('y(t)')
xlabel('Time [s]')
print -depsc Step_Gs_P12.eps
-----
```

